

Cross-Sectional Properties of Structural Members

Resultant of Parallel Forces

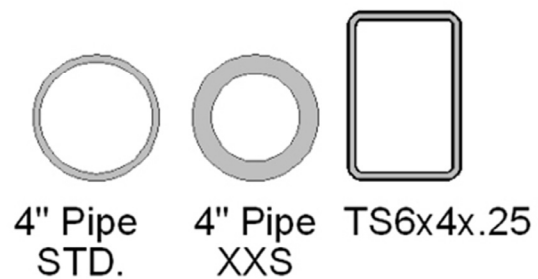
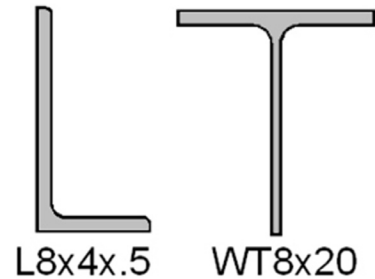
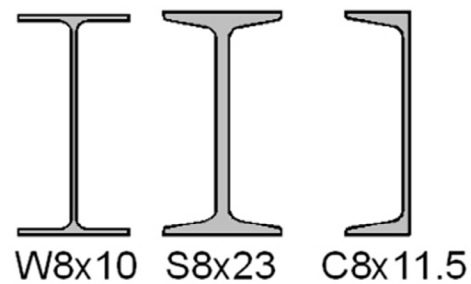
Center of Gravity

Centroid of Area

First Moment of Area

Second Moment of Area
(Moment of Inertia)

Radius of Gyration



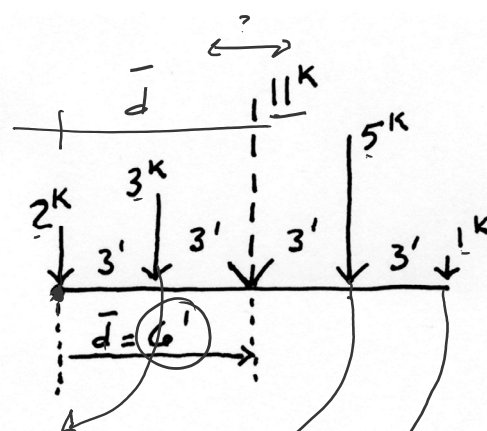
Parallel Force Resultant

The resultant is the single force that has the same effect as the group of forces.

$$\sum M = \sum (\underline{\mathbf{F}} \times \underline{d}) = \underline{\mathbf{R}} \times \underline{\bar{d}}$$

$$\sum \underline{\mathbf{F}} = \underline{\mathbf{R}}$$

$$\underline{\bar{d}} = \frac{\sum (\underline{\mathbf{F}} \times \underline{d})}{\sum \underline{\mathbf{F}}} = \frac{0 + 9 + 45 + 12}{2 + 3 + 5 + 1} = \frac{66}{11} = 6'$$



Centers

The point about which a body may be balanced.

This is the point of application of the resultant weight.

Center of Gravity

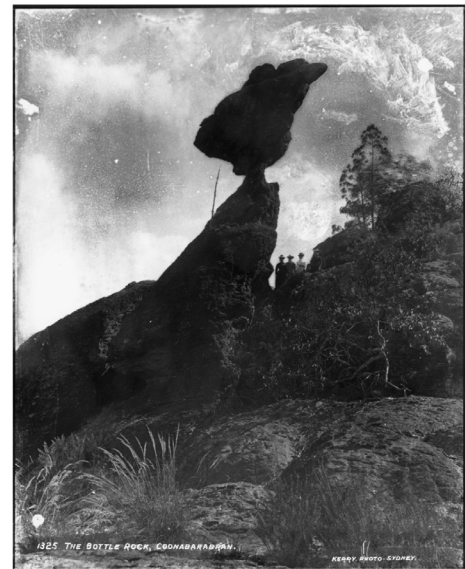
$$\bar{x} = \frac{\sum W \times d_x}{\sum W}$$

Center of Volume

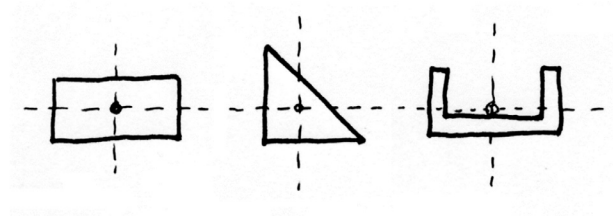
$$\bar{x} = \frac{\sum V \times d_x}{\sum V}$$

Center of Area (centroid)

$$\bar{x} = \frac{\sum A \times d_x}{\sum A}$$



Tyrrell Photographic Collection, Powerhouse Museum



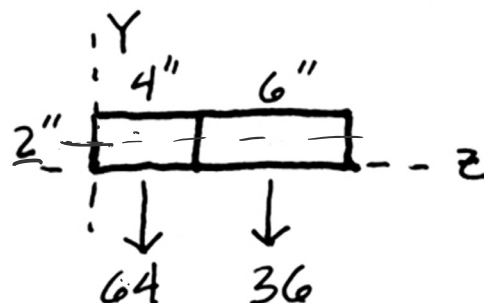
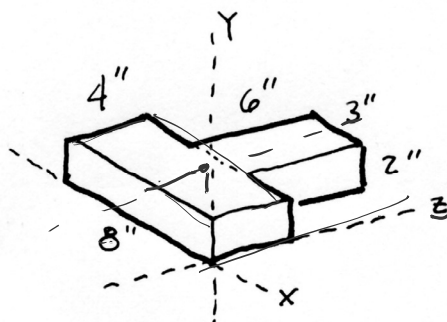
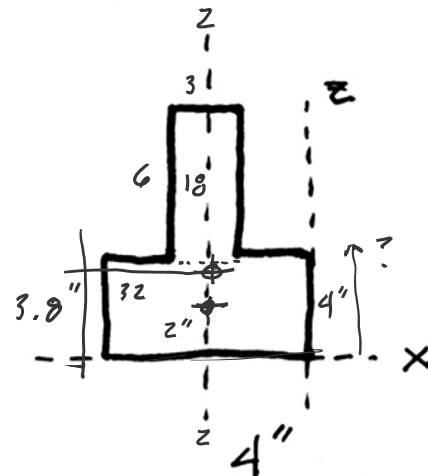
Center of Gravity (or Volume)

The Center of Gravity is located at the point defined by:

$(\bar{x}, \bar{y}, \bar{z})$

$(\bar{x}, \bar{y}, \bar{z})$

$$\bar{x} = \frac{\sum W \times d_x}{\sum W} = \frac{(64 \times 2) + (36 \times 7)}{64 + 36} = \frac{380}{100}$$



Center of Area - the Centroid

The “center of area” for a cross section.

$$\bar{x} = \frac{\sum (\text{Area} \times d_x)}{\sum \text{Area}} = \frac{A x_A + B x_B + C x_C}{A + B + C}$$

$$\text{Area}_A = 2 \times 7 = 14$$

$$\text{Area}_B = 3 \times 2 = 6$$

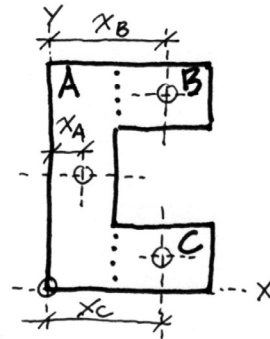
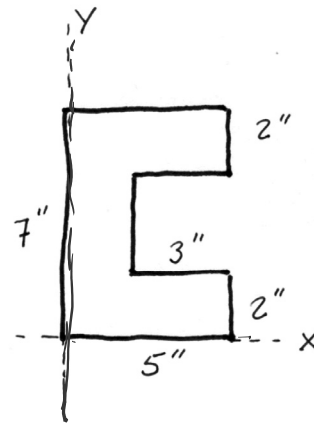
$$\text{Area}_C = 3 \times 2 = 6$$

$$\text{sum} = 26$$

$$x_A = 1$$

$$x_B = 3.5$$

$$x_C = 3.5$$



Centroid Example 1 cont.

$$\text{Area}_A = 2 \times 7 = 14$$

$$x_A = 1$$

$$\text{Area}_B = 3 \times 2 = 6$$

$$x_B = 3.5$$

$$\text{Area}_C = 3 \times 2 = 6$$

$$x_C = 3.5$$

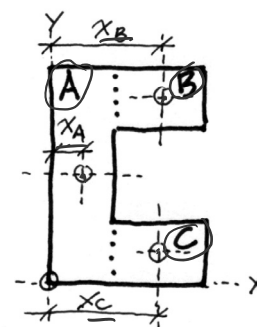
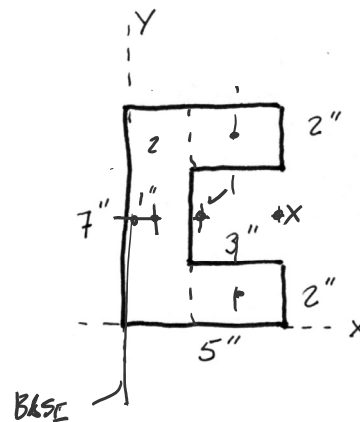
$$\text{sum} = 26$$

Calculation.

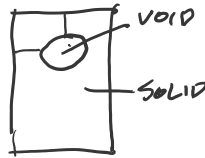
$$\bar{x} = \frac{\sum \text{Area} \times d_x}{\sum \text{Area}} = \frac{A x_A + B x_B + C x_C}{A + B + C}$$

$$\bar{x} = \frac{(14 \times 1) + (6 \times 3.5) + (6 \times 3.5)}{14 + 6 + 6}$$

$$\bar{x} = \frac{56}{26} = \underline{\underline{2.15''}}$$



Centroid Example 1 cont.

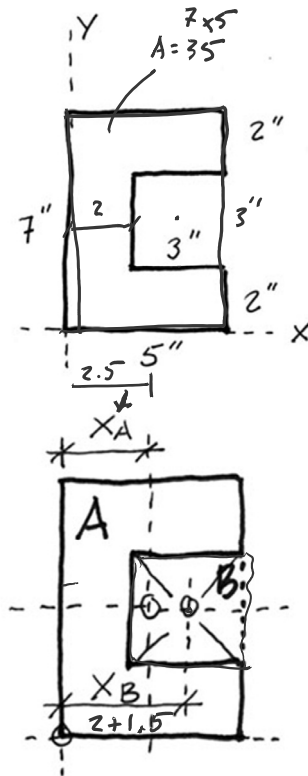


Calculation: by Solid - Void.

$$\bar{x} = \frac{\sum A \times d_x}{\sum A} = \frac{A x_A - B x_B}{A - B}$$

$$\bar{x} = \frac{\sum (35 \times 2.5) - (9 \times 3.5)}{\sum 35 - 9} = \frac{56}{26}$$

$$\bar{x} = 2.15"$$



Static Moment of Area

1st moment of area

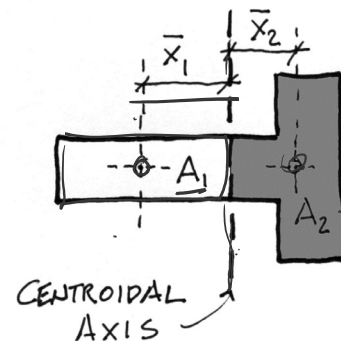
$$f_v = \frac{VQ}{Ib}$$

The tendency of an area alone to rotate about an axis in the plane of that area.

$$Q = A \bar{x}$$

At the Neutral Axis

$$A_1 \bar{x}_1 = A_2 \bar{x}_2$$

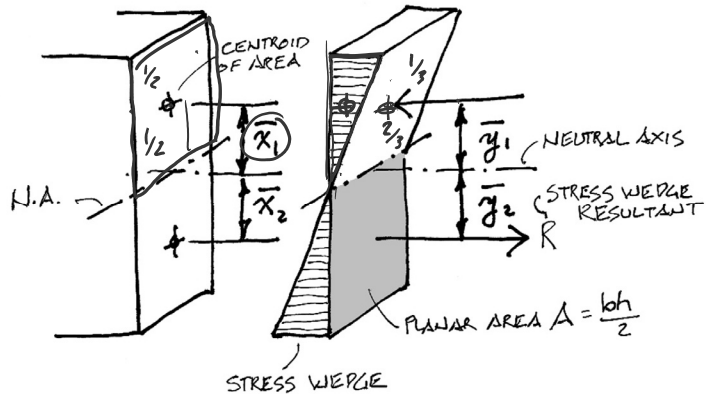


Moment of Inertia

2nd moment of area

By definition:

$$I_x = A \bar{x} \bar{y}$$



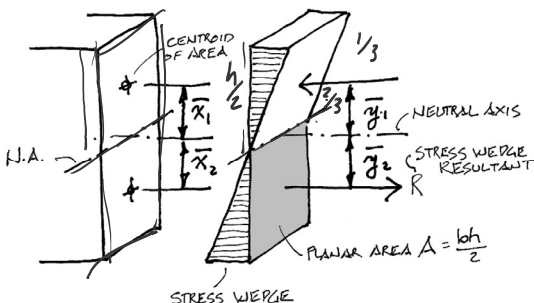
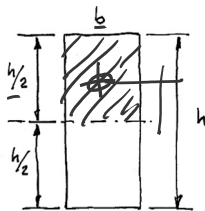
For a rectangle at the N.A.

$$I_x = \frac{bh^3}{12}$$

Moment of Inertia

2nd moment of area

$$I_x = A \bar{x} \bar{y}$$



FOR A RECTANGULAR SECTION:

$$\begin{aligned} \checkmark A &= b \left(\frac{h}{2} \right) \\ \checkmark \bar{x} &= \frac{h}{2} \div 2 = \frac{h}{4} \\ \checkmark \bar{y} &= \frac{2}{3} \frac{h}{2} = \frac{h}{3} \end{aligned}$$

$$A \bar{x} \bar{y}_{(TOP)} = \frac{bh}{2} \frac{h}{4} \frac{h}{3} = \frac{bh^3}{24}$$

$$A \bar{x} \bar{y}_{(BOTTOM)} = \frac{bh}{2} \frac{h}{4} \frac{h}{3} = \frac{bh^3}{24}$$

FOR TOTAL SECTION:

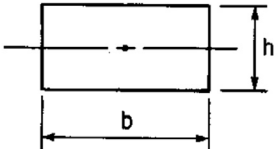
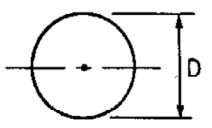
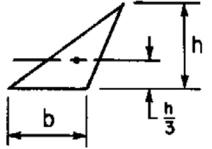
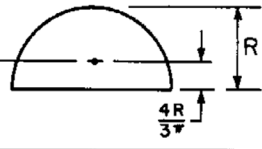
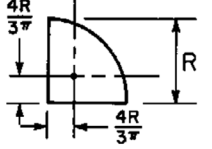
$$2 \times \frac{bh^3}{24} = \frac{bh^3}{12}$$

RECTANGLE

$$I_x = \frac{bh^3}{12}$$

Moment of Inertia

Solutions for basic shapes:

	Shape	Moment of inertia
Rectangle		$I = \frac{1}{12}bh^3$
Circle		$I = \frac{\pi D^4}{64} = \frac{\pi R^4}{4}$
Triangle		$I = \frac{1}{36}bh^3$
Semicircle		$I = \left(\frac{\pi}{8} - \frac{8}{9\pi} \right) R^4 = 0.11R^4$
Quarter circle		$I = \left(\frac{\pi}{8} - \frac{8}{9\pi} \right) \frac{R^4}{2} = 0.055R^4$

Moment of Inertia

Solutions for basic shapes:

- Single Shapes
- Combination Shapes

COMBINATION SECTIONS

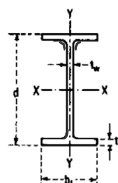
S shapes and channels

Properties of sections

The diagram shows an S-shape section with a vertical centerline. A horizontal dashed line represents the X-X axis, passing through the centroid. A vertical dashed line represents the Y-Y axis, also passing through the centroid. The distance from the X-X axis to the top flange is labeled y₂. The distance from the X-X axis to the bottom flange is labeled y₁. The distance from the Y-Y axis to the outer edge of the flange is labeled y-bar.

Beam	Channel	Total Wt. per Ft	Total Area	AXIS X-X					AXIS Y-Y				
				<i>I</i>	<i>S</i> ₁ = <i>I</i> / <i>y</i> ₁	<i>S</i> ₂ = <i>I</i> / <i>y</i> ₂	<i>r</i>	<i>y</i> ₁	<i>I</i>	<i>S</i>	<i>r</i>	<i>r_T</i>	
				in. ⁴	in. ³	in. ³	in.	in.	in. ⁴	in. ³	in.		
S 10 × 25.4	C 8 × 11.5	36.9	10.84	176	27.2	46.6	4.02	6.45	39.4	9.85	1.91	2.44	
× 25.4	C 10 × 15.3	40.7	11.95	186	27.6	52.9	3.94	6.73	74.2	14.8	2.49	3.16	
S 12 × 31.8	C 8 × 11.5	43.3	12.73	299	39.8	63.2	4.84	7.50	42.0	10.5	1.82	2.38	
× 31.8	C 10 × 15.3	47.1	13.84	316	40.4	71.4	4.78	7.82	76.8	15.4	2.36	3.06	
× 40.8	C 10 × 15.3	56.1	16.49	377	50.1	80.0	4.78	7.53	81.0	16.2	2.22	2.94	

WIDE FLANGE SHAPES

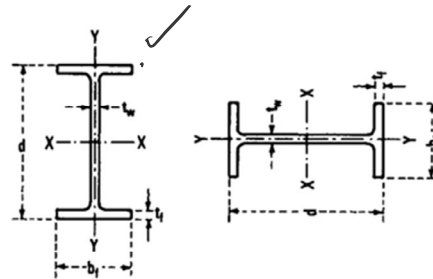


Theoretical Dimensions and Properties for Designing

Theoretical Dimensions and Properties for Designing														
Section Number	Weight per Foot	Area of Section	Depth of Section	Flange			Web Thickness	Axis X-X			Axis Y-Y			r_T
				Width	Thickness			I_x	S_x	r_x	I_y	S_y	r_y	
lb	A	d	b_f	t_f	t_w	in. ⁴	in. ³	in.	in. ⁴	in. ³	in.	in.		
W27 x 178	178	52.3	27.81	14.085	1.190	0.725	6990	502	11.8	555	78.8	3.26	3.72	
161		47.4	27.59	14.020	1.080	0.660	6280	455	11.5	497	70.9	3.24	3.70	
146		42.9	27.38	13.965	0.975	0.605	5630	411	11.4	443	63.5	3.21	3.68	
W27 x 114	114	33.5	27.29	10.070	0.930	0.570	4090	299	11.0	159	31.5	2.18	2.58	
102		30.0	27.09	10.015	0.830	0.515	3620	267	11.0	139	27.8	2.15	2.56	
94		27.7	26.92	9.990	0.745	0.490	3270	243	10.9	124	24.8	2.12	2.53	
84		24.8	26.71	9.960	0.640	0.460	2850	213	10.7	106	21.2	2.07	2.49	

Section Properties

WIDE FLANGE SHAPES



Theoretical Dimensions and Properties for **Designing**

STRONG

WEAK

Section Number	Weight per Foot	Area of Section	Depth of Section	Flange		Web Thickness	Axis X-X			Axis Y-Y			r _T
				Width	Thick-ness		I _x	S _x	r _x	I _y	S _y	r _y	
PCF	(lb)	in. ²	in.	in.	in.	in.	in. ⁴	in. ³	in.	in. ⁴	in. ³	in.	in.
W27 x <i>deep</i>	178	52.3	27.81	14.085	1.190	0.725	6990	502	11.6	555	78.8	3.26	3.72
	161	47.4	27.59	14.020	1.080	0.660	6280	455	11.5	497	70.9	3.24	3.70
	146	42.9	27.38	13.965	0.975	0.605	5630	411	11.4	443	63.5	3.21	3.68
W27 x	114	33.5	27.29	10.070	0.930	0.570	4090	299	11.0	159	31.5	2.18	2.58
	102	30.0	27.09	10.015	0.830	0.515	3620	267	11.0	139	27.8	2.15	2.56
	94	27.7	26.92	9.990	0.745	0.490	3270	243	10.9	124	24.8	2.12	2.53
	84	24.8	26.71	9.960	0.640	0.460	2850	213	10.7	106	21.2	2.07	2.49

Section Properties

PROPERTIES OF SAWN LUMBER SECTIONS

Rectangular :

$$A = bd$$

$$I = db^3/12$$

$$S = I/c_{max}$$

$$c = d/2 \text{ (maximum)}$$

$$f_b = \frac{Mc}{I} = \frac{M}{S}$$

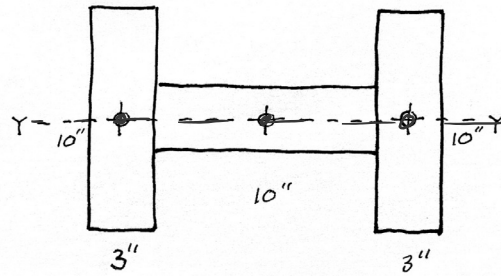


Nominal Size b × d	Actual Size b × d	Area in. ²	I _x in. ⁴	S _x in. ³
1 × 4	3/4 × 3 1/2	2.63	2.68	1.53
1 × 6	" × 5 1/2	4.13	10.40	3.78
1 × 8	" × 7 1/4	5.44	23.82	6.57
1 × 10	" × 9 1/4	6.94	49.47	10.70
1 × 12	" × 11 1/4	8.44	88.99	15.83
2 × 4	1 1/2 × 3 1/2	5.25	5.36	3.06
2 × 6	" × 5 1/2	8.25	20.80	7.56
2 × 8	" × 7 1/4	10.88	47.64	13.14
2 × 10	" × 9 1/4	13.88	98.93	21.39
2 × 12	" × 11 1/4	16.88	177.98	31.64
3 × 4	2 1/2 × 3 1/2	8.75	8.93	5.10
3 × 6	" × 5 1/2	13.75	34.66	12.60
3 × 8	" × 7 1/4	18.13	79.39	21.90
3 × 10	" × 9 1/4	23.13	164.89	35.65
3 × 12	" × 11 1/4	28.13	296.63	52.73
4 × 4	3 1/2 × 3 1/2	12.25	12.50	7.15
4 × 6	" × 5 1/2	19.25	48.53	17.65
4 × 8	" × 7 1/4	25.38	111.15	30.66
4 × 10	" × 9 1/4	32.38	230.84	49.91
4 × 12	" × 11 1/4	39.38	415.28	73.83

Moment of Inertia

Shapes with common centroidal axes

$$I_{\text{solid}} + I_{\text{solid}} = I_x$$



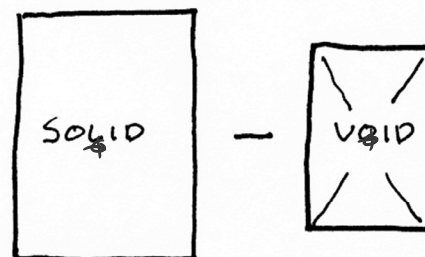
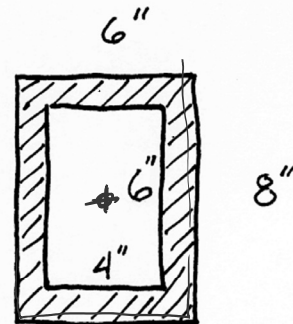
$$\frac{3(10)^3}{12} + \frac{10(3)^3}{12} + \frac{3(10)^3}{12}$$

$$250 \text{ in}^4 + 22.5 \text{ in}^4 + 250 \text{ in}^4 = 522.5 \text{ in}^4$$

Moment of Inertia

Shapes with common centroidal axes

$$I_{\text{solid}} - I_{\text{void}} = I_x$$



$$\frac{6 \times 8^3}{12} - \frac{4 \times 6^3}{12}$$

$$256 - 72 = 184 \text{ in}^4$$

Moment of Inertia

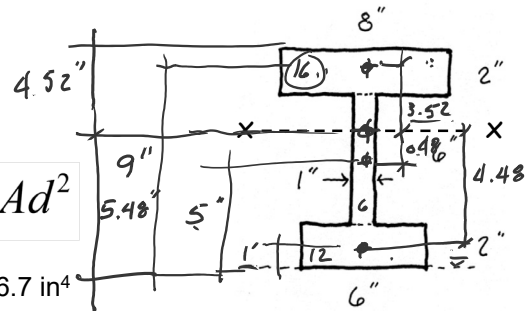
The **Transfer Equation** or **Parallel Axis Theorem**,
taken about the x-x axis:

$$\bar{y} = \frac{\sum Ay}{\sum A}$$

$$\bar{y} = 186/34 = 5.48"$$

$$I_x = \sum \bar{I}_x + \sum Ad^2$$

$$I_x = 27.3 + 439.4 = 466.7 \text{ in}^4$$



Shape	A	y	Ay	\bar{I}_x	d, in.	Ad^2
<div style="text-align: center;"> <div style="width: 80px; height: 20px; border: 1px solid black; margin: 0 auto;"></div> <div style="text-align: center;">8"</div> </div>	(2)(8) = 16	9	144	$(\frac{1}{12})(8)(2)^3 = 5.3$	3.52	$(16)(3.52)^2 = 198$
<div style="text-align: center;"> <div style="width: 20px; height: 60px; border: 1px solid black; margin: 0 auto;"></div> <div style="text-align: center;">1"</div> </div>	(1)(6) = 6	5	30	$(\frac{1}{12})(1)(6)^3 = 18$	0.48	$6(0.48)^2 = 1.4$
<div style="text-align: center;"> <div style="width: 80px; height: 20px; border: 1px solid black; margin: 0 auto;"></div> <div style="text-align: center;">2"</div> </div>	(2)(6) = 12	1	12	$(\frac{1}{12})(6)(2)^3 = 4$	4.48	$12(4.48)^2 = 240$
	$\sum A = 34$	$\sum Ay = 186$		$\sum \bar{I}_x = 27.3$		$\sum Ad^2 = 439.4$

$$\bar{y} = 186/34 = 5.48"$$

$$I_x = 27.3 + 439.4 = 466.7 \text{ in}^4$$

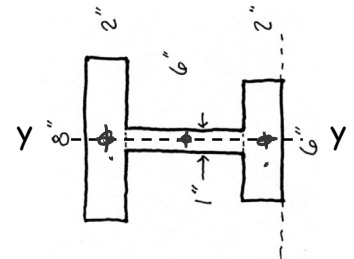
Moment of Inertia

The **Transfer Equation** or **Parallel Axis Theorem**:

$$I_y = \sum \bar{I}_y + \sum Ad^2$$

Taken about the y-y axis:

$$I_y = 121.8 + 0 = 121.8$$



Shape	A	\bar{I}_y	d	Ad^2
<div style="text-align: center;"> <div style="width: 80px; height: 20px; border: 1px solid black; margin: 0 auto;"></div> <div style="text-align: center;">8"</div> </div>	16	$(\frac{1}{12})(2)(8)^3 = 85.3$	0	0
<div style="text-align: center;"> <div style="width: 20px; height: 60px; border: 1px solid black; margin: 0 auto;"></div> <div style="text-align: center;">1"</div> </div>	6	$(\frac{1}{12})(6)(1)^3 = 0.5$	0	0
<div style="text-align: center;"> <div style="width: 80px; height: 20px; border: 1px solid black; margin: 0 auto;"></div> <div style="text-align: center;">2"</div> </div>	12	$(\frac{1}{12})(2)(6)^3 = 36.0$	0	0
		$\sum \bar{I}_y = 121.8$		0

SUMMARY:

$$I_x = 466.7 \text{ in}^4$$

$$I_y = 121.8 \text{ in}^4$$

Radius of Gyration

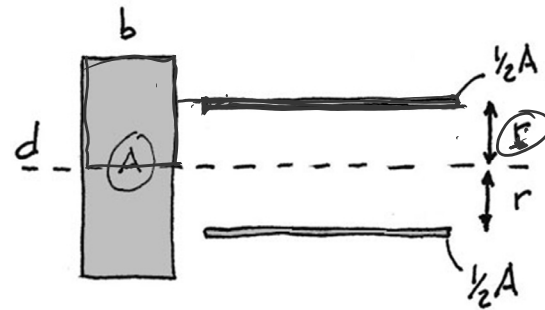
The distance from the centroid where all area could be collected to yield an equivalent Moment of Inertia.

$$I = A r^2$$

$$r = \sqrt{\frac{I}{A}}$$

$$r = 0.289 d$$

for a rectangle about the N.A



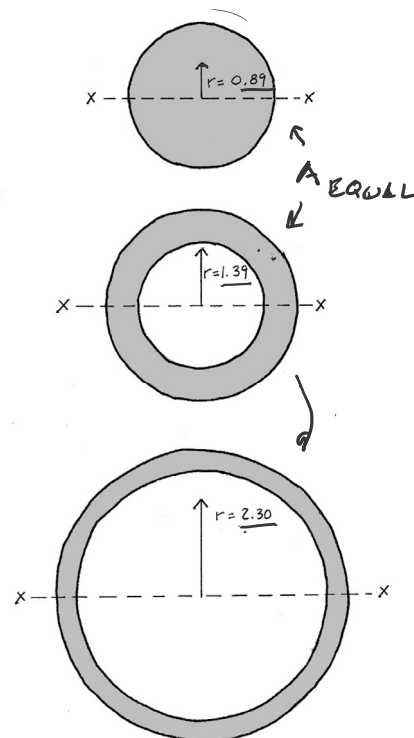
Radius of Gyration

The larger the radius of gyration, the more resistant the section is to buckling.

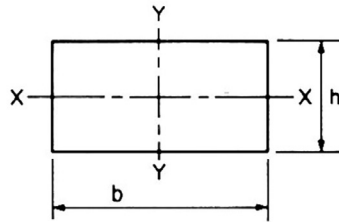
Area below is a constant, while diameter increases.

OD	ID	t	A	r
3.57	0.00	1.78	10.00	0.89
3.71	1.00	1.35	10.00	0.96
4.09	2.00	1.05	10.00	1.14
4.66	3.00	0.83	10.00	1.39
5.36	4.00	0.68	10.00	1.67
6.14	5.00	0.57	10.00	1.98
6.98	6.00	0.49	10.00	2.30
7.86	7.00	0.43	10.00	2.63
8.76	8.00	0.38	10.00	2.97
9.68	9.00	0.34	10.00	3.30
10.62	10.00	0.31	10.00	3.65

$$P_{cr} = \frac{\pi^2 E}{(KL/r)^2}$$



Section Formulas



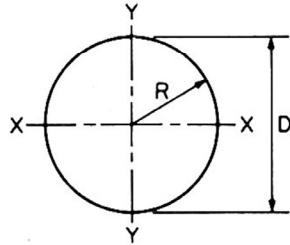
Rectangle.

Rectangle

$$A = bh,$$

$$I_x = \frac{1}{12}bh^3,$$

$$r_x = \sqrt{I_x/A} = \underline{0.288h}.$$



Circle.

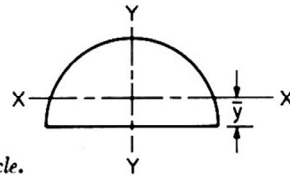
Circle

$$A = \frac{1}{4}\pi D^2 = \pi R^2,$$

$$I_x = \frac{\pi D^4}{64} = \frac{\pi R^4}{4},$$

$$r_x = \sqrt{I_x/A} = \frac{D}{4} = \frac{R}{2},$$

$$J = I_x + I_y = \frac{\pi D^4}{32} = \frac{\pi R^4}{2}.$$



Semicircle.

Semicircle

$$A = \frac{1}{8}\pi D^2 = \frac{1}{2}\pi R^2,$$

$$\bar{y} = \frac{4R}{3\pi},$$

$$I_x = 0.00682D^4 = 0.11R^4,$$

$$I_y = \frac{\pi D^4}{128} = \frac{\pi R^4}{8},$$

$$r_x = 0.264R.$$